\overline{U}_p = 2sN = mean piston speed s = stroke, N = revolutions/time

w_{i,g} = gross indicated work per cycle = indicated work considering only compression and expansion

w_{i,n} = indicated net work per cycle

P_{i,g} = gross indicated power, P_f = power to overcome friction + pumping losses, P_b = net power at the crank

 $P_b = P_{i,g} - P_f = \tau \omega = 2\pi N \tau$ $\tau = torque, \omega = rotational speed in rad/time$

 $mep = w_{net}/V_d$ $w_{net} = net work per cycle$, $V_d = displacement volume$

imep = (indicated net work)/V_d

bmep = $P_b/(V_d N/n)$ n = number of crank revolutions per cycle τ = bmep $V_d/(2\pi n)$ so max τ at max bmep

fmep = friction mean effective pressure = imep - bmep

 $\eta_m = P_b/P_{i,g}$ = mechanical efficiency (Caution: Some books use $P_{i,net}$)

 η_c = combustion efficiency = (amount of fuel burned)/(amount of fuel delivered)

bsfc = brake specific fuel consumption = $\dot{m}_{\rm f}/P_{\rm b}$

 n_{fc} = fuel conversion efficiency = $P_b/(\dot{m}_f Q_{LHV})$ Q_{LHV} = Lower heating value of fuel used as approximation

 η_v = volumetric efficiency = $\dot{m}_a/(\rho_a V_d N/n)$ ρ_a can be atmospheric or intake air density, must be specified

A/F =
$$\dot{m}_a/\dot{m}_f = 1/(F/A)$$
 $\phi = (F/A)_{actual}/(F/A)_{stoic} = equivalence ratio = 1/\lambda$

$$\begin{split} P_{b} &= \eta_{fc} \dot{m}_{f} \, Q_{LHV} = \dot{m}_{a} \, (F/A) \, \eta_{fc} \, Q_{LHV} = \eta_{v} \, \eta_{fc} \, \rho_{a} \, V_{d} \, (F/A) \, Q_{LHV} \, N/n \end{split} \tag{F/A} optimized for particular operating point. \\ \tau &= \eta_{v} \, \eta_{fc} \, \rho_{a} \, V_{d} \, (F/A) \, Q_{LHV} \, / (2\pi \, n) \end{aligned}$$
 No explicit dependence on engine speed! τ tracks η_{v} .

 $\frac{T_0}{T} = 1 + \frac{k-1}{2} M^2 \qquad \qquad \frac{P_0}{p} = \left[1 + \frac{k-1}{2} M^2\right]^{\frac{k}{(k-1)}} \qquad \text{Ram air:} \qquad \frac{P_{b2}}{P_{b1}} = \left[1 + \frac{k-1}{2} M^2\right]^{\frac{k+1}{2(k-1)}} \qquad \text{where } P_b = \text{power}$ $\dot{m} = \rho \vec{V} A = \frac{P_0}{\sqrt{T_0}} AM \sqrt{\frac{k}{R}} \left[1 + \frac{k-1}{2} M^2\right]^{-\left[\frac{k+1}{2(k-1)}\right]} = \rho_0 c_0 A \sqrt{\frac{2}{k-1} \left[\left(\frac{P}{P_0}\right)^{\frac{2}{k}} - \left(\frac{P}{P_0}\right)^{\frac{k+1}{k}}\right]} \qquad \text{Zero subscripts indicate stagnation cond.}$

for choked flow $\dot{m} = \rho_0 c_0 A \left[\frac{k+1}{2} \right]^{-\left[\frac{k+1}{2(k-1)} \right]}$

 $\frac{P_{b2}}{P_{b1}} = \frac{P_2 - P_{\nu 2}}{P_1 - P_{\nu 1}} \sqrt{\frac{T_1}{T_2}}$ Power correction for pressure, temperature and humidity.